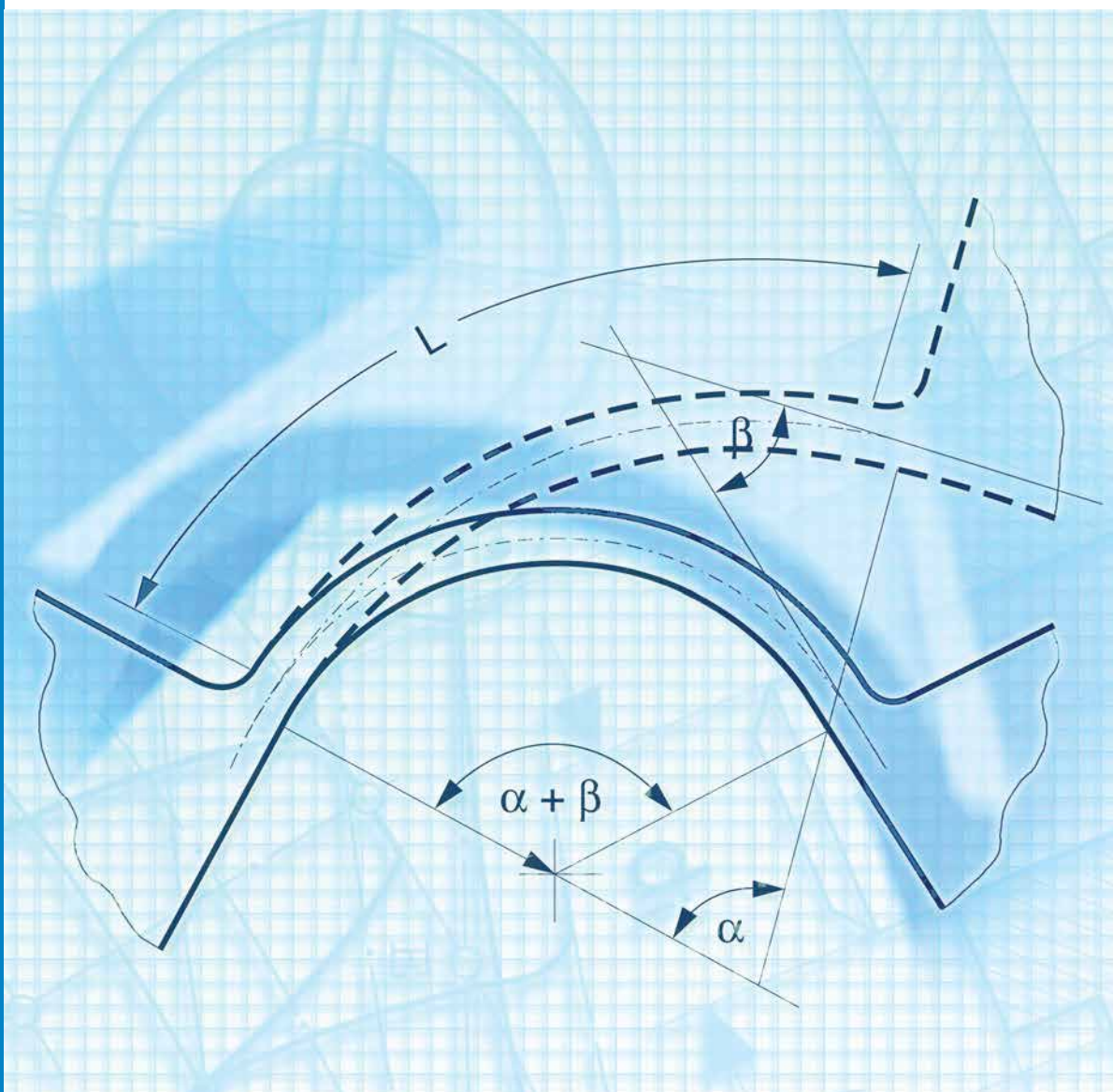


Integral hinges in engineering plastics



COPYRIGHT:

All rights reserved, in particular for reproduction and copying, and for distribution as well as for translation. No part of this publication may be reproduced or processed by means of electronic systems, reproduced or distributed (by photocopying, microfilm or any other process), without written permission by Ticona.

© 2004 Ticona GmbH, Kelsterbach

NOTICE TO USERS:

To the best of our knowledge, the information contained in this publication is accurate, however we do not assume any liability whatsoever for the accuracy and completeness of such information. The information contained in this publication should not be construed as a promise or guarantee of specific properties of our products.

Further, the analysis techniques included in this publication are often simplifications and, therefore, approximate in nature. More vigorous analysis techniques and prototype testing are strongly recommended to verify satisfactory part performance. Anyone intending to rely on any recommendation or to use any equipment, processing technique or material mentioned in this publication should satisfy themselves that they can meet all applicable safety and health standards.

It is the sole responsibility of the users to investigate whether any existing patents are infringed by the use of the materials mentioned in this publication.

Properties of molded parts can be influenced by a wide variety of factors including, but not limited to, material selection, additives, part design, processing conditions and environmental exposure. Any determination of the suitability of a particular material and part design for any use contemplated by the user is the sole responsibility of the user. The user must verify that the material, as subsequently processed, meets the requirements of the particular product or use. The user is encouraged to test prototypes or samples of the product under the harshest conditions to be encountered to determine the suitability of the materials.

Material data and values included in this publication are either based on testing of laboratory test specimens and represent data that fall within the normal range of properties for natural material or were extracted from various published sources. All are believed to be representative. These values alone do not represent a sufficient basis for any part design and are not intended for use in establishing maximum, minimum, or ranges of values for specification purposes. Colorants or other additives may cause significant variations in data values.

We strongly recommend that users seek and adhere to the manufacturer's current instructions for handling each material they use, and to entrust the handling of such material to adequately trained personnel only. Please call the numbers listed for additional technical information. Call Customer Services at the number listed for the appropriate Material Safety Data Sheets (MSDS) before attempting to process our products. Moreover, there is a need to reduce human exposure to many materials to the lowest practical limits in view of possible adverse effects. To the extent that any hazards may have been mentioned in this publication, we neither suggest nor guarantee that such hazards are the only ones that exist.

The products mentioned herein are not intended for use in medical or dental implants.

Ticona GmbH

Information Service

Tel. +49 (0) 180-584 2662 (Germany)

+49 (0) 69-305 16299 (Europe)

Fax +49 (0) 180-202 1202 (Germany and Europe)

e-mail infoservice@ticona.de

Internet www.ticona.com

Contents

1. <i>Introduction</i>	3	7. <i>Calculation examples</i>	8
2. <i>Requirements for integral hinges</i>	3	7.1 Rocker switch on a cassette recorder	8
3. <i>Method of manufacture</i>	4	7.2 Electric connector for motor vehicles	9
4. <i>Materials and material modifications</i>	4	8. <i>Typical applications</i>	10
5. <i>Designing integral hinges</i>	5	8.1 Fastening device for greenhouse shading	10
5.1 Integral hinges without post-mould flexing	5	8.2 Video cassette box	11
5.1.1 High flex numbers required	5	8.3 Coffee maker housing	12
5.1.2 Integral hinges as assembly aids	5	8.4 Filter housing for a washing machine	13
5.2 Integral hinges with post-mould flexing	6	8.5 Cardan mounting	14
6. <i>Injection moulding of components with integral hinges</i>	6	8.6 Transmission head of an electric razor	15
6.1 Processing conditions	7	8.7 Sewing machine box	16
6.2 Gate design and location	7	8.8 Toaster housing	17
		9. <i>Explanation of symbols</i>	18
		10. <i>Literature</i>	18

®Hostaform

acetal copolymer (POM)

®Celanex

polybutylene terephthalate (PBT)

®Hostacom

reinforced polypropylene (PP)

®Hostalen PP

polypropylene (PP)

®Hostalen

polyethylene (PE)

® = registered trademark

1. Introduction

Various designs of integral hinge are used for many different applications but they all utilize a principle that depends on some typical properties of plastics: high toughness, ductility and flexural fatigue strength. The number of flexes an integral hinge can withstand varies between one and several million depending on requirements.

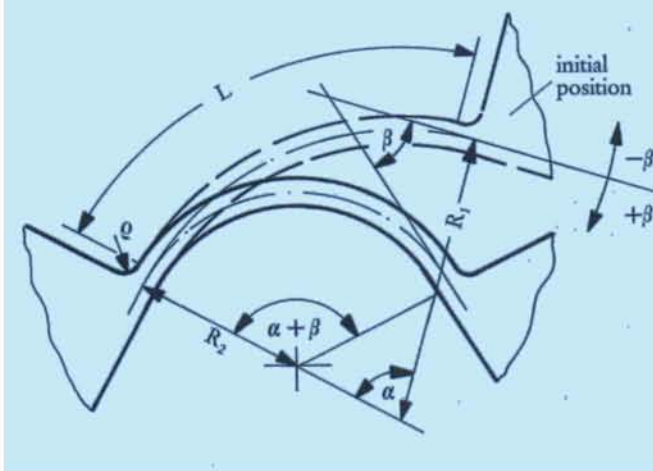
Integral hinges are flexible connections between two movable parts made from thermoplastics. The complete assembly is produced in one piece from the same plastic without additional connecting points and is classified as a spring joint [1]. In these joints designed as flexural spring elements, the aim is generally to achieve the lowest possible restoring force (fig. 1).

Integral hinges have no mutually sliding surfaces. They are thus wear-free with low internal friction. Their disadvantages are the limited loadbearing capacity of the joint due to low hinge thickness h and the dependence – common to all thermoplastics – of mechanical properties on time and temperature.

2. Requirements for integral hinges

Requirements for integral hinges vary according to the number of flexes required N , the flex angle β and the flex frequency f . Whereas, for example, in the mounting of the cutting head for an electric razor high flex numbers N with a low flex angle β and high flex frequency f are required, integral hinges are often used as assembly aids where the hinge needs to be flexed only once or a few times through a relatively large angle. For this purpose, high material toughness – even at low temperatures – is generally required.

Fig. 1: Illustration of integral hinge dimensions



3. *Method of manufacture*

Components with integral hinges are normally injection moulded. In addition, extrusion blow moulding has acquired some considerable importance in the production of double-walled boxes with integral hinges for sewing machines, measuring instruments, tools etc. [2]. It is also technically possible to extrude profiles with integral hinges, e. g. for glazing gaskets. Generally speaking, this process produces the final shape of the integral hinge. By subsequent plastic deformation of the hinge in an embossing operation, the loadbearing capacity, maximum permissible flex angle and maximum permissible number of flexes can be increased. Within certain limits, this increase is higher the higher the degree of stretching λ (= ratio of original thickness h_0 of the hinge to the thickness after coining h) in the hinge area. The maximum achievable degree of stretching depends on the embossing conditions, particularly on temperature and deformation rate, and varies according to the particular thermoplastic. Polypropylene is particularly suitable because it undergoes a structural transformation between about 80 and 140 °C [3] which brings a corresponding improvement in hinge properties. This characteristic means that – assuming correct hinge design – the structural transformation produced on flexing the hinge for the first time after injection moulding is sufficient to increase the loadbearing capacity of the hinge significantly.

4. *Materials and material modifications*

For integral hinges required to withstand high flex numbers, materials with good fatigue properties are suitable. Partially crystalline thermoplastics have better fatigue properties than amorphous thermoplastics. A good indication of fatigue behaviour is provided by fatigue strength. This is the relationship determined in the fatigue test (DIN 53 442) between stress amplitude σ_a and deformation amplitude ε_a and the number of cycles to failure N of the test specimen. Among the Hoechst AG thermoplastics, the following exhibit good fatigue behaviour:

- ®Hostalen (polyethylene)
- ®Hostalen PP (polypropylene)
- ®Hostacom (reinforced polypropylene)
- ®Hostaform (acetal copolymer)
- ®Celanex (polybutylene terephthalate)
- ®Vandar
(impact-modified polybutylene terephthalate).

Within each product range, high-molecular-weight grades with a narrow molecular weight distribution have better fatigue properties. On the other hand, melt flowability is limited with these products. It is therefore necessary to test in each individual case whether a particular component can be completely filled in injection moulding despite the integral hinge acting as a flow obstacle.

The incorporation of fillers and reinforcing materials generally results in inferior hinge properties, i. e. poorer fatigue behaviour and reduced ductility as toughness declines. In this respect, spherical fillers (glass microspheres, calcium carbonate, barium sulphate) and fillers with a foliated structure (talc) have a less adverse effect than fibrous reinforcing materials (glass fibres, carbon fibres). High permissible deformation amplitudes ε_a with high flex numbers N can be achieved by elastomer modification of the base material. Elastomer additions also improve hinge properties in formulations containing fillers and reinforcing materials.

5. Designing integral hinges

5.1 Integral hinges without post-mould flexing

5.1.1 High flex numbers required

These integral hinges are dimensioned on the basis of the Wöhler curves $\sigma_a = f(N)$, fig. 2 obtained in the fatigue test. The deformation amplitude can be assigned by calculation to the stress amplitude σ_a .

From $\sigma = \varepsilon \cdot E$ and $\sigma = \frac{M}{W}$ where $W = \frac{b \cdot h^2}{6}$ we obtain

$$\varepsilon_a = \frac{\sigma_a}{E_s} = \frac{M_b}{W \cdot E_s} = \frac{6 \cdot M_b}{b \cdot h^2 \cdot E_s} \quad (1)$$

ε_a	deformation amplitude (strain)
σ_a	stress amplitude
E_s	secant modulus from the tensile test
M_b	bending moment
W	section modulus for rectangular cross section $= \frac{b \cdot h^2}{6}$
b	width of the integral hinge
h	thickness of the integral hinge

In table 1, the deformation amplitudes obtained for flex cycle numbers $N = 10^6$ and $N = 10^7$ are shown.

For a given flex angle β (see fig. 1) and required flex number N , the hinge length L and hinge thickness h must be selected to ensure that the deformation amplitude ε_a obtained at N is not exceeded, i.e. the

$$\text{outer fibre elongation } \varepsilon_b \leq \varepsilon_a(N) \quad (2)$$

Assuming that the integral hinge is circular and that the circular shape is retained in flexing, then only the radius of curvature R of the circular arc changes. The radius of curvature R_1 of the hinge with starting angle α (fig. 1) is calculated as follows:

$$R_1 = \frac{L}{\alpha} \quad (3)$$

α measured in radians (rad)

Conversion from α° into α (rad):

$$360^\circ = 6.28 \text{ rad}$$

$$1^\circ = 0.01745 \text{ rad}$$

$$\alpha = 0.01745 \cdot \alpha^\circ$$

After flexure of the hinge through the angle $\pm \beta$ then

$$\text{either } R_2 = \frac{L}{\alpha + \beta} \text{ oder } R_2 = \frac{L}{\alpha - \beta} \quad (4)$$

α, β measured in radians (rad)

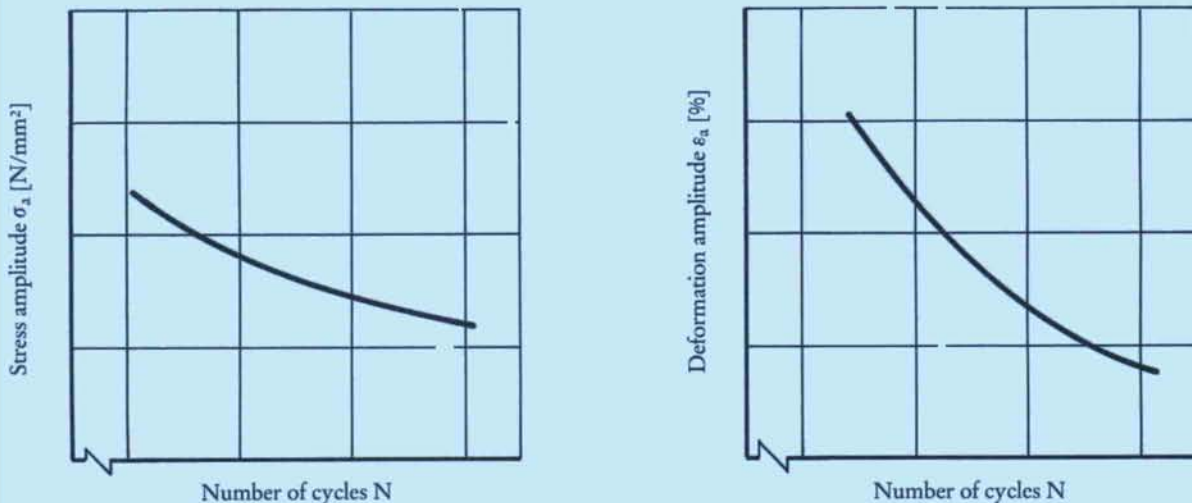
Between the outer fibre elongation ε_b and radius of curvature R the following equation applies:

$$\varepsilon_b = \frac{h}{2} \left(\frac{1}{R_1} - \frac{1}{R_2} \right), \text{ ie} \quad (5)$$

$$\text{for } +\beta \quad \varepsilon_b = \frac{h}{2} \left(\frac{\alpha}{L} - \frac{\alpha + \beta}{L} \right) \cdot 100\% \quad (6a)$$

$$\text{for } -\beta \quad \varepsilon_b = \frac{h}{2} \left(\frac{\alpha}{L} - \frac{\alpha - \beta}{L} \right) \cdot 100\% \quad (6b)$$

Fig 2: Wöhler curve and outer fibre elongation curve from the fatigue test (schematic)



For $R_2 < R_1$, we obtain negative values in brackets in equation 5. In further calculation, the absolute (ie unsigned) values of ε_b should be inserted.

The values quoted in table 1 for ε_a can be used as the basis for integral hinge design (after allowing a safety factor of $S = 1.1$ to 1.2):

$$\varepsilon_b = \frac{\varepsilon_a (N)}{S} \quad (7)$$

Special attention should be given to the fillet radius ϱ at the transition between the moulding and integral hinge. The notch effect at this point can be reduced by suitable radiusing ("streamlining") (see C.3.3 Design of mouldings made from engineering plastics).

5.1.2 Integral hinges as assembly aids

Reliable dimensioning is ensured if the deformation ε_b occurring in the outer fibres does not exceed the deformation at yield stress:

$$\varepsilon_b \leq \varepsilon_S \quad (8)$$

Frequently, however, because of restricted space, greater deformation of the hinge must be expected.

Then it is important to ensure that the deformation ε_b is smaller than the elongation at break ε_R , see table 2.

$$\varepsilon_S < \varepsilon_b < \varepsilon_R \quad (9)$$

In these cases, however, stress whitening may occur in the hinge area.

5.2 Integral hinges with post-mould flexing

An exact correlation between the degree of stretch achieved λ in embossing and hinge properties (maximum permissible outer fibre deformation ε_b , maximum permissible number of cycles N) is not yet possible. For this reason, integral hinges required to withstand a high number of flex cycles should be dimensioned for the "elastic" deformation range, $\varepsilon_b \leq \varepsilon_S$, according to section 5.1.2.

Table 1: Stress amplitude σ_a and deformation amplitude ε_a (N) in the flexural fatigue range for Hostaform, Celanex, Hostalen PP, Hostacom and Hostalen

Material	N = 10 ⁶		N = 10 ⁷	
	σ_a [N/mm ²]	ε_a [%]	σ_a [N/mm ²]	ε_a [%]
Hostaform C 2521	46	2.6	34	1.7
Hostaform C 9021	40	2.1	28	1.2
Hostaform C 13021	37	2.0	26	1.1
Hostaform C 27021	34	1.5	19	0.75
Hostaform S 9063	48	4.0	39	3.0
Hostaform S 9064	33	3.0	26	2.0
Hostaform S 27076	21	4.0	19	3.0
Hostaform C 9021 GV 1/30	58	0.7	50	0.6
Celanex 2500	48	2.1	29	1.2
Celanex 2300 GV 1/30	35	0.5	30	0.3
Hostalen PPR 1042	28	2.7	24	2.1
Hostacom M4 N01	41	1.5	32	1.0
Hostacom G3 N01	32	0.65	27	0.5
Hostalen GM 5010 T3			21	2.3
Hostalen GF 7750			18	1.3
Hostalen GC 7260			6	0.5

For integral hinges which are to be flexed only once, the recommendations in section 5.1.2 apply. Structural transformation by once-only flexion should be confined to integral hinges made from polypropylene and its modifications. In this respect, PP copolymers behave more favourably than PP homopolymers. The tendency to white fracture is less with random copolymers than with block copolymers.

Table 2: Deformation at yield stress ε_S and elongation at break ε_R of Hostaform, Celanex, Hostalen PP, Hostacom and Hostalen

Material	ε_S [%]	ε_R [%]
Hostaform C 2521	9	35
Hostaform C 9021	9	28
Hostaform C 13021	9	25
Hostaform C 27021	9	20
Hostaform S 9063	9	60
Hostaform S 9064	9	90
Hostaform S 27076	9	> 150
Hostaform C 9021 GV 1/30	—	3
Celanex 2500	4	15
Celanex 2300 GV 1/30	—	2.5
Hostalen PPR 1042	13	> 400
Hostacom M4 N01	5	5
Hostacom G2 N01	10	50
Hostacom G3 N01	—	3
Hostalen GM 5010 T3	10	> 200
Hostalen GF 7750	10	> 200
Hostalen GC 7260	10	> 200

6. Injection moulding of components with integral hinges

6.1 Processing conditions

The integral hinge acts as a flow obstacle to the melt. The lower the melt viscosity, the more easily this obstacle can be overcome. For this reason, high melt and mould temperatures are an advantage. Since the viscosity of pseudoplastic polymer melts decreases with increasing shear rate, the injection rate should be as fast as possible. A sufficiently high mould temperature prevents over-rapid freezing of the melt in the hinge area. As a result, the hold-on pressure is able to be effective even in the mould section behind the hinge; in this way, sink marks can be avoided.

6.2 Gate design and location

The gate designs normally used in injection moulding may also be employed for components with integral hinges. Good hinge properties as well as high flex numbers can be achieved if the melt front reaches the hinge, if possible, at the same time across its full width and flows through it evenly and without delay. For this purpose, film gates or pinpoint gates, fig 3 are suitable. Similar conditions apply if a single gate is located at a sufficient distance from the hinge, fig. 4. As a general requirement, the mould must be gated in such a way as to prevent:

- local melt stagnation and resultant undercooling of the melt
- weld lines in the integral hinge
- trapped air in the integral hinge.

Weld lines occur when a gate is provided in both parts of the moulding separated by the integral hinge, fig. 5. In such cases, it is important to ensure through suitable dimensioning of the runner and/or gate cross sections that the weld line lies outside the integral hinge.

Fig. 3: Uniform mould filling with a film gate (left) and multiple pinpoint gate (right)

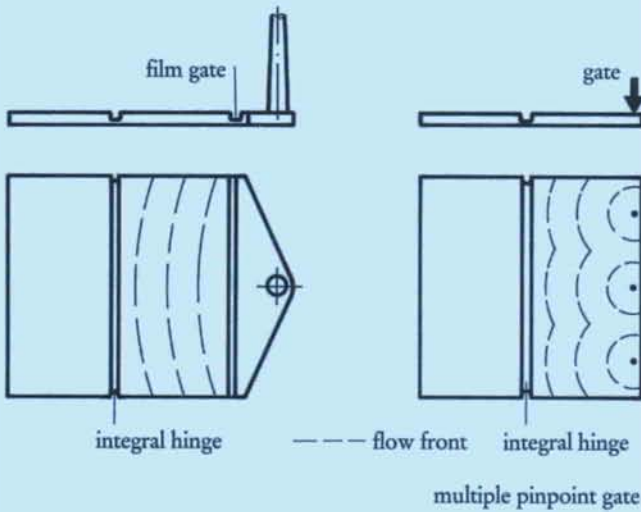


Fig. 4: Melt front approximately parallel to the integral hinge – achieved by locating the tunnel gate at a sufficient distance from the integral hinge

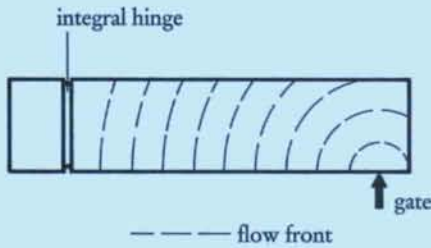
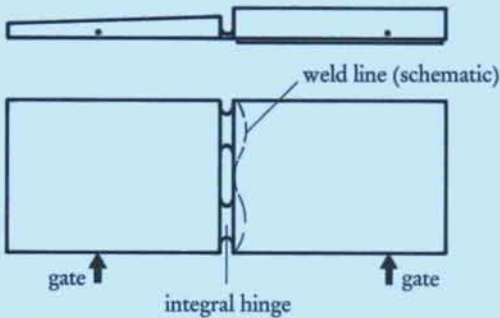


Fig. 5: Two tunnel gates: weld line outside the integral hinge owing to the different volumes of the lid and box

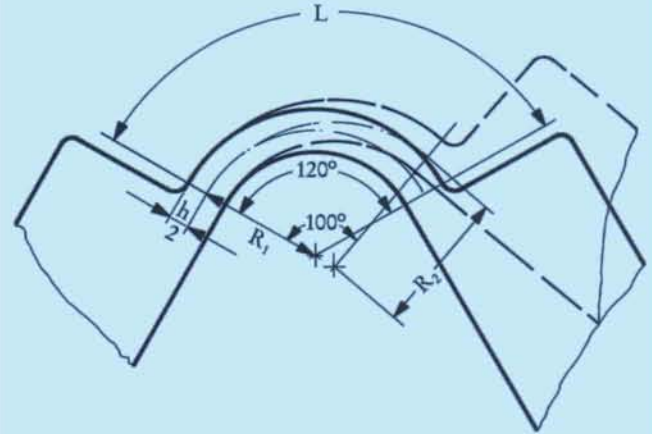


7. Calculation examples

7.1 Rocker switch on a cassette recorder

In a rocker switch for a cassette recorder made from Hostaform C 13021, the two sides of the switch are movably connected at an angle of $\alpha = 120^\circ$ by an integral hinge, fig. 6.

Fig. 6: Integral hinge on a rocker switch



When the switch is operated, this angle is reduced to 100° (flex angle $\beta = 20^\circ$). The aim is to find the dimensions of the integral hinge, which must withstand at least 10^6 flex cycles. According to table 1, for Hostaform C 13021 with $N = 10^6$

$$\begin{aligned}\varepsilon_b &= \frac{\varepsilon_a}{S} \\ &= \frac{2\%}{1.1} \\ &= 1.8\%\end{aligned}$$

is permissible.

The selected hinge length is $L = 5 \text{ mm}$ and the hinge thickness $h = 0.5 \text{ mm}$. The radius of curvature R_1 of the unflexed hinge can be calculated as:

$$\begin{aligned}R_1 &= \frac{L}{\alpha} \quad \alpha = 120^\circ = 0.01745 \cdot 120 = 2.094 \text{ rad} \\ &= \frac{5 \text{ mm}}{2.094} \\ &= 2.388 \text{ mm}\end{aligned}$$

For the radius of curvature of the deformed hinge

$$\begin{aligned}R_2 &= \frac{L}{\alpha - \beta} \quad \alpha - \beta = 100^\circ = 1.745 \text{ rad} \\ &= \frac{5 \text{ mm}}{1.745} \\ &= 2.865 \text{ mm}\end{aligned}$$

With this, the outer fibre elongation ε_b can be calculated according to equation (5)

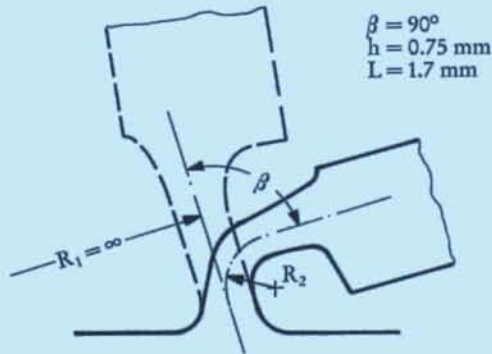
$$\begin{aligned}\varepsilon_b &= \frac{h}{2} \left(\frac{1}{R_1} - \frac{1}{R_2} \right) \cdot 100\% \\ &= \frac{0.5}{2} \left(\frac{1}{2.388} - \frac{1}{2.865} \right) \cdot 100\% \\ &= \underline{\underline{1.74\%}}\end{aligned}$$

The occurring deformation ε_b is thus less than the permissible deformation of 1.8%.

7.2 Electric connector for motor vehicles

On an electric connector made from Hostacom G2 N01, a part movably joined with an integral hinge is bent through approx. 90°, after fixing on the electrical contacts, and then locked in position with a nondetachable snapfit, fig. 7.

Fig. 7: Integral hinge on an electric connector for a motor vehicle



Dimensions of the integral hinge: $h = 0.75 \text{ mm}$
 $L = 1.5 \text{ mm}$
 $R_1 = \infty$
 $\frac{1}{R_1} = \frac{1}{\infty} = 0$

The radius of curvature of the deformed hinge is calculated as follows:

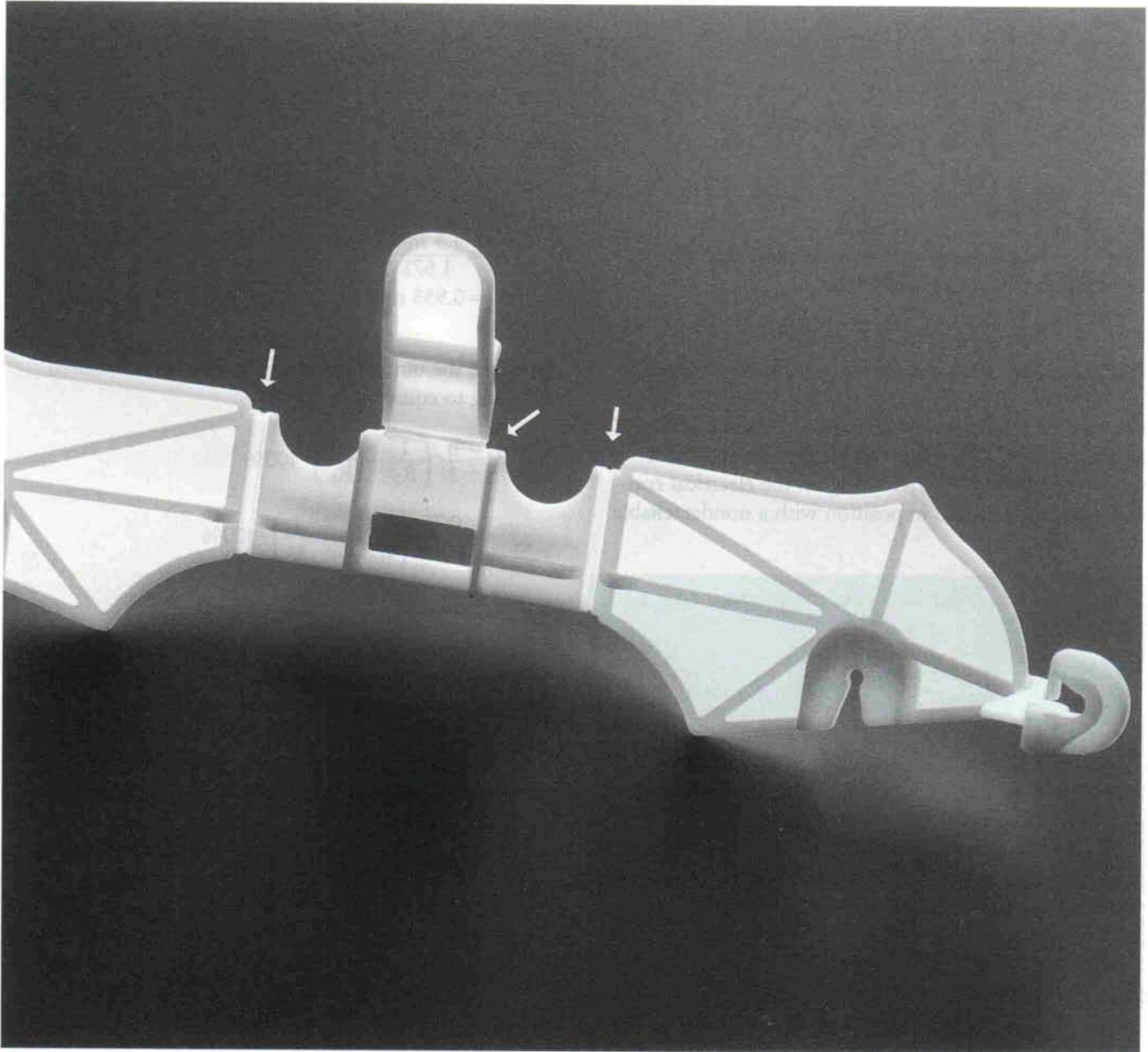
$$\begin{aligned}R_2 &= \frac{L}{\beta} & \beta &= 90^\circ = 1.571 \text{ rad} \\ &= \frac{1.5 \text{ mm}}{1.571} \\ &= 0.955 \text{ mm}\end{aligned}$$

With this, the outer fibre elongation can be calculated according to equation (5)

$$\begin{aligned}\varepsilon_b &= \frac{h}{2} \left(\frac{1}{R_2} - \frac{1}{R_1} \right) \cdot 100\% \\ &= \frac{0.75}{2} \left(\frac{1}{0.955} - \frac{1}{\infty} \right) \cdot 100\% \\ &= \underline{\underline{39.3\%}}\end{aligned}$$

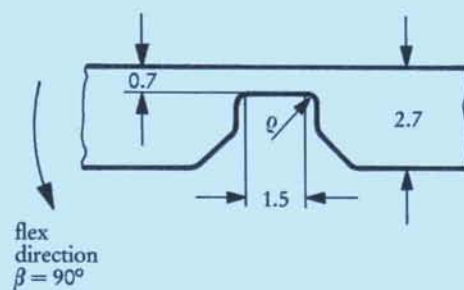
The outer fibre elongation occurring is less than the elongation at break of $\varepsilon_R = 50\%$ quoted in table 2 for Hostacom G2 N01.

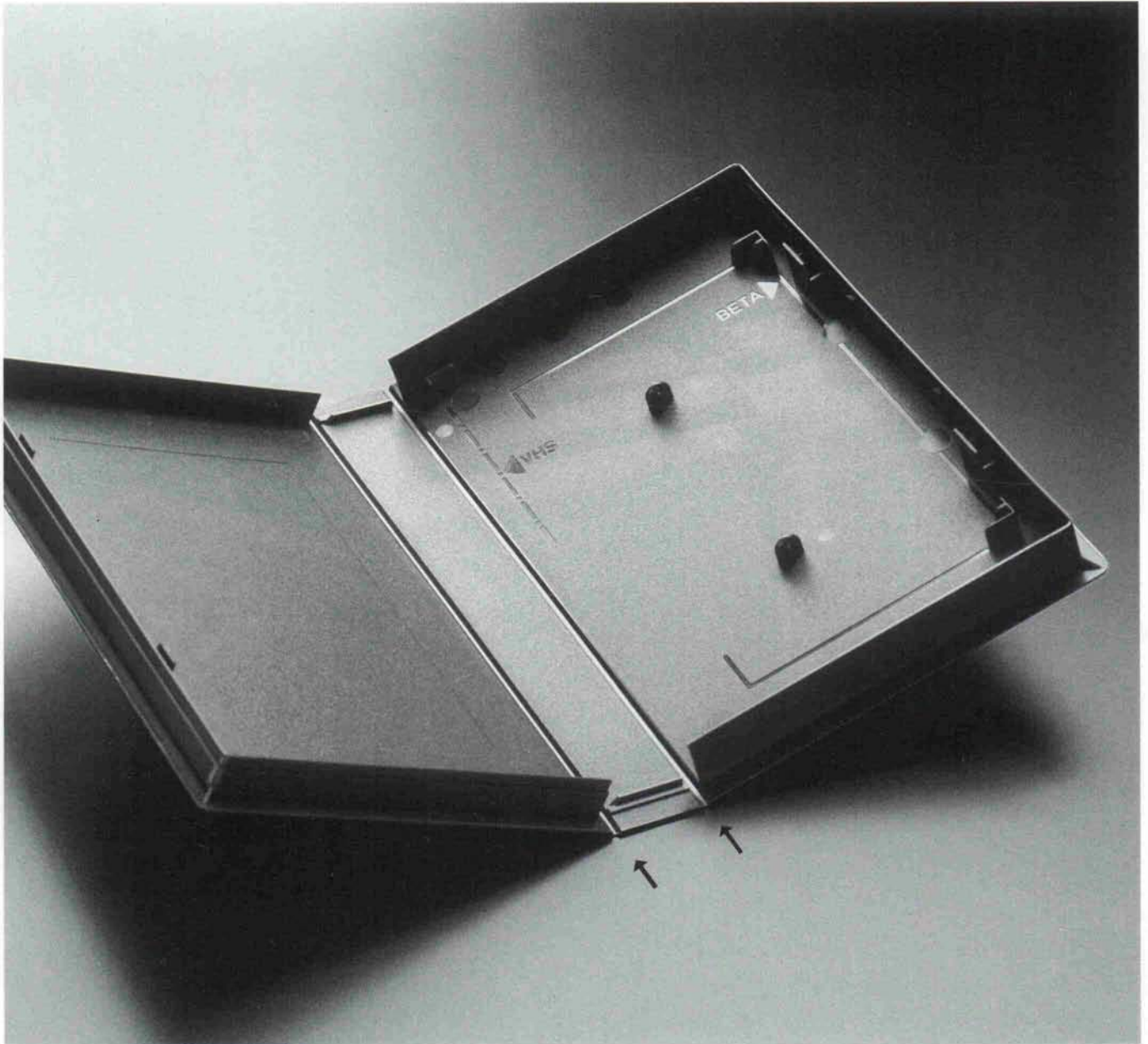
8. Typical applications



8.1 Fastening device for greenhouse shading

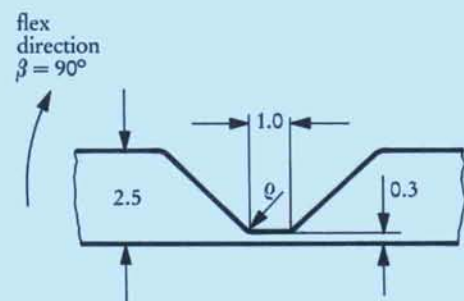
This fastening device with three integral hinges is produced from Hostaform S 9064. The thickness of the integral hinges is $h = 0.7$ mm and the length is $L = 1.5$ mm. The flex angle is $\beta = 90^\circ$. The moulding is centrally gated via a pinpoint gate.

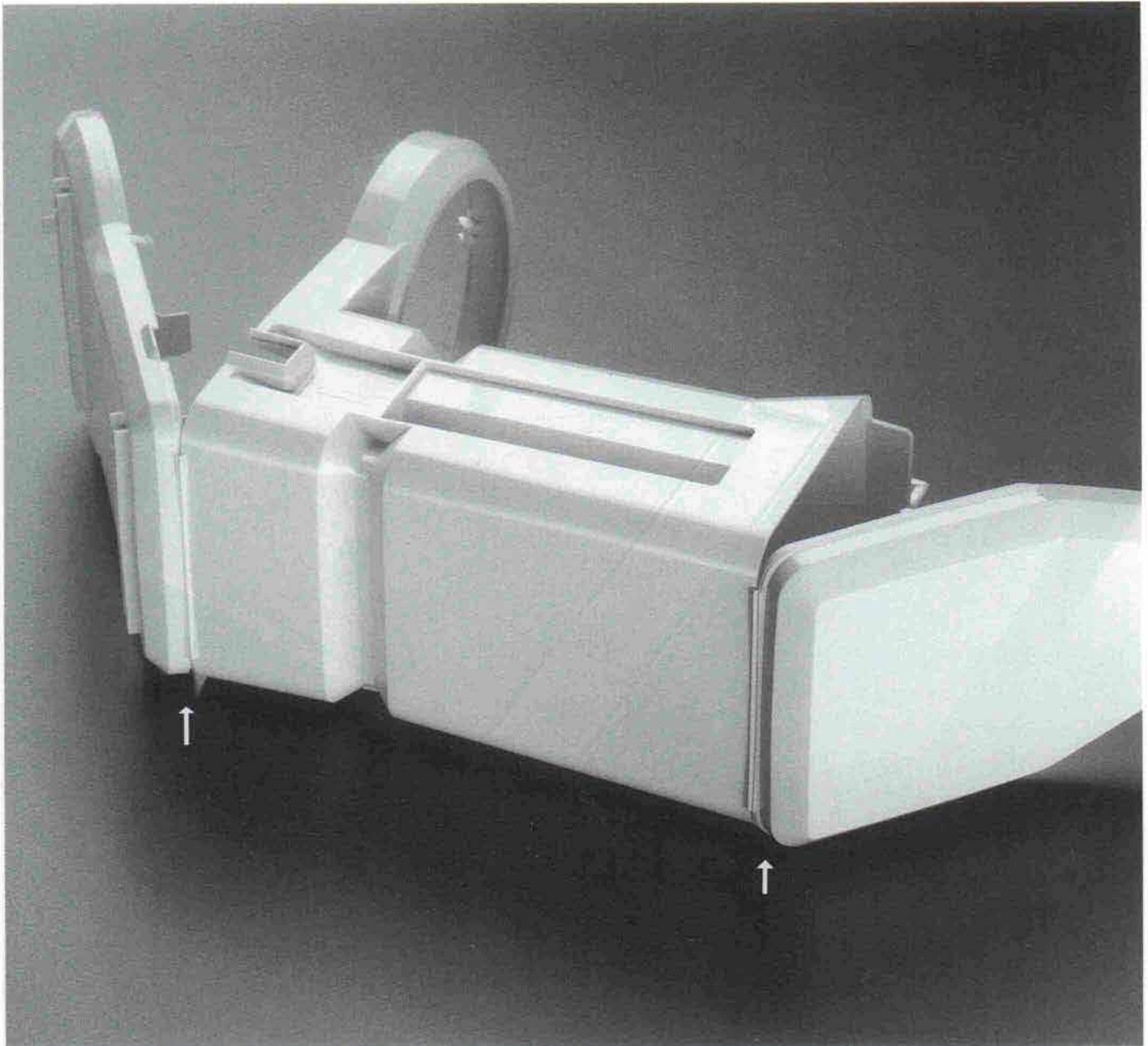




8.2 Video cassette box

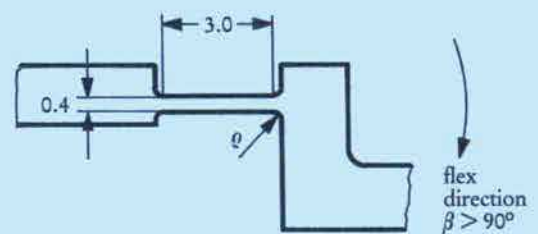
This box is injection moulded from the easy-flowing, antistatic-modified copolymer Hostalen PPW 1752 S 1 ASTL. The moulding is gated in the centre via two pinpoint gates. There are similar designs with just one central pinpoint gate.

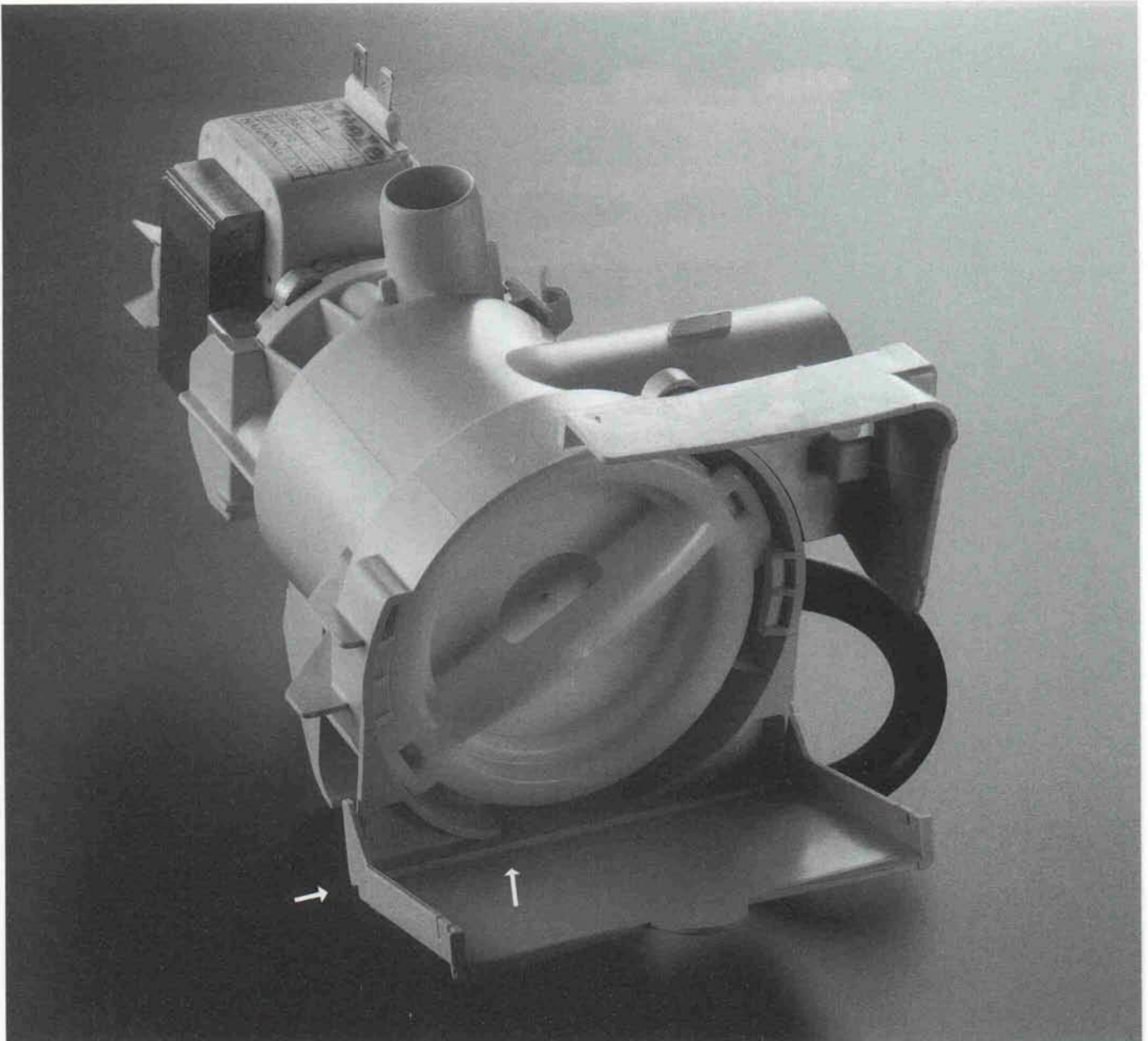




8.3 Coffee maker housing

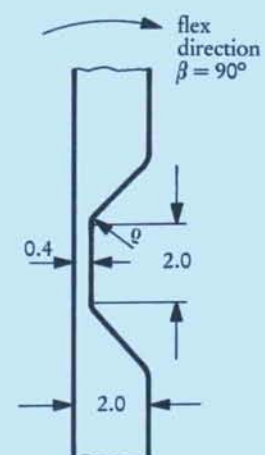
In this example, the lid and base are connected via integral hinges to the main housing. The materials used are Hostacom M1 U01 and unreinforced polypropylene. While the integral hinge with the base is only flexed during assembly and in the possible event of repair, the lid hinge is flexed every time the coffee maker is used.

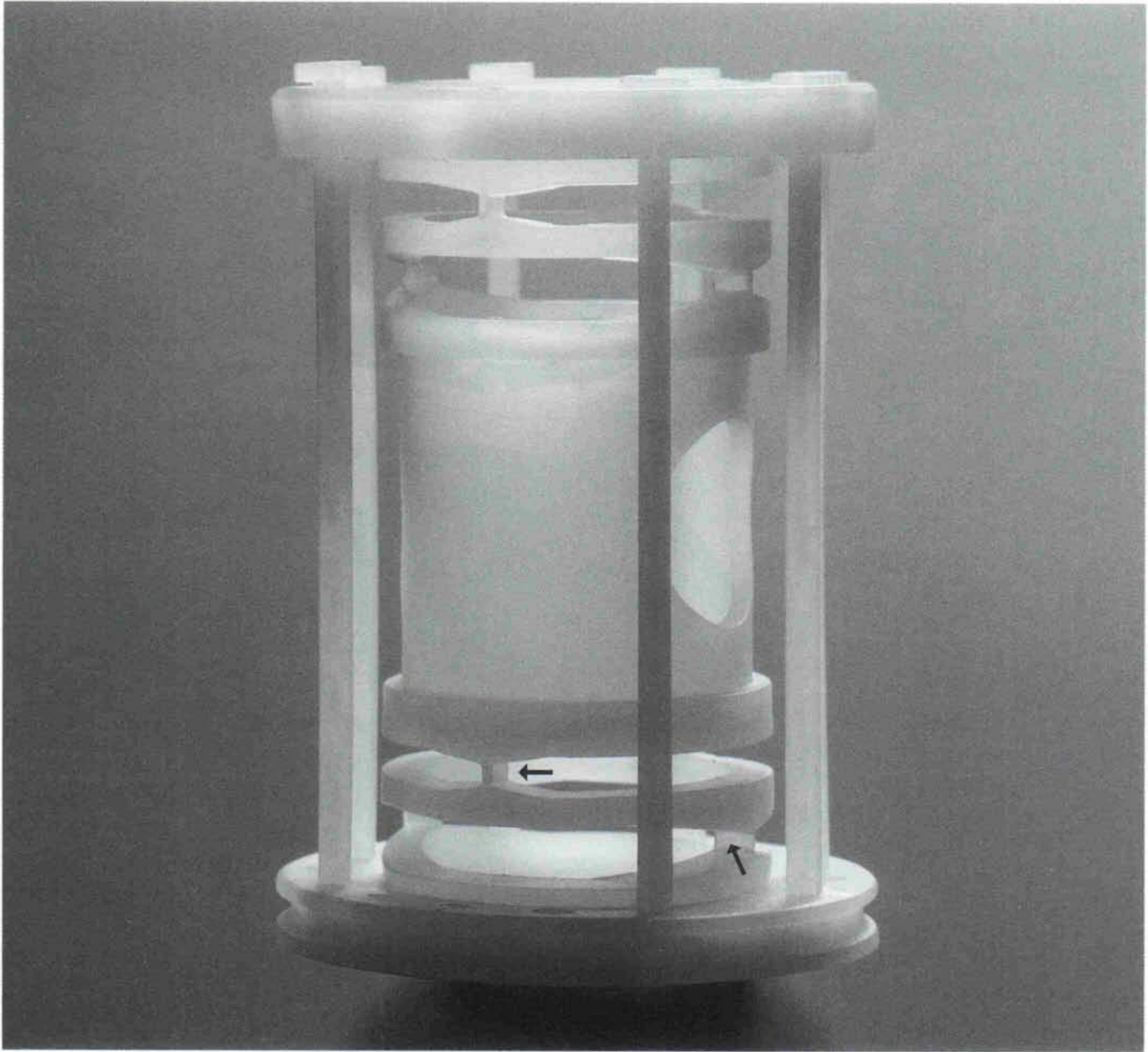




8.4 Filter housing for a washing machine

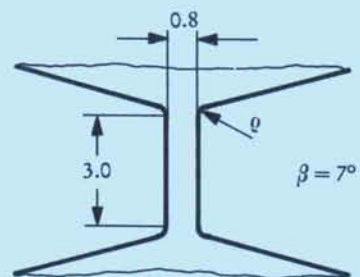
This filter housing is made from Hostacom M4 N01. To clean the filter, the integrally hinged flap is pulled down. This enables any residual water to be collected without any problem. The cylindrical section of the housing is gated via four tunnel gates.

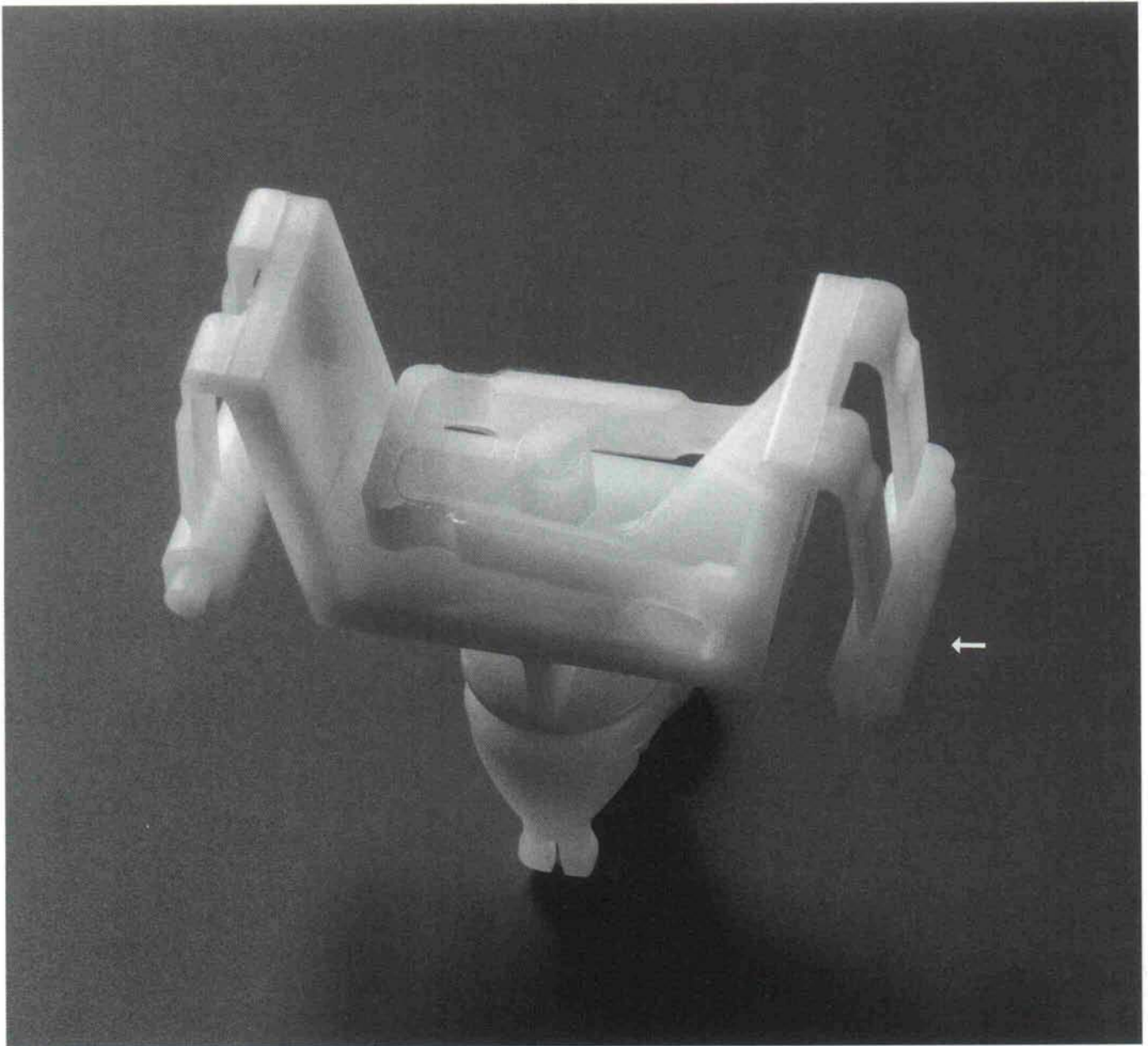




8.5 Cardan mounting

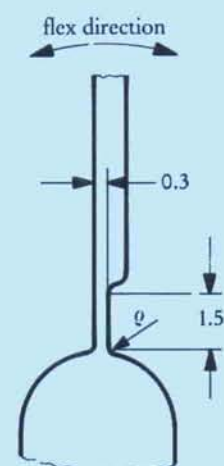
This Cardan mounting for the baseplate of an orbital sander is made from Hostalen PPR 1042. A total of eight integral hinges ensure the mobility of the baseplate parallel to the housing in all directions. The integral hinges are stressed by high flex numbers and oscillating frequency with relatively small flex angles.

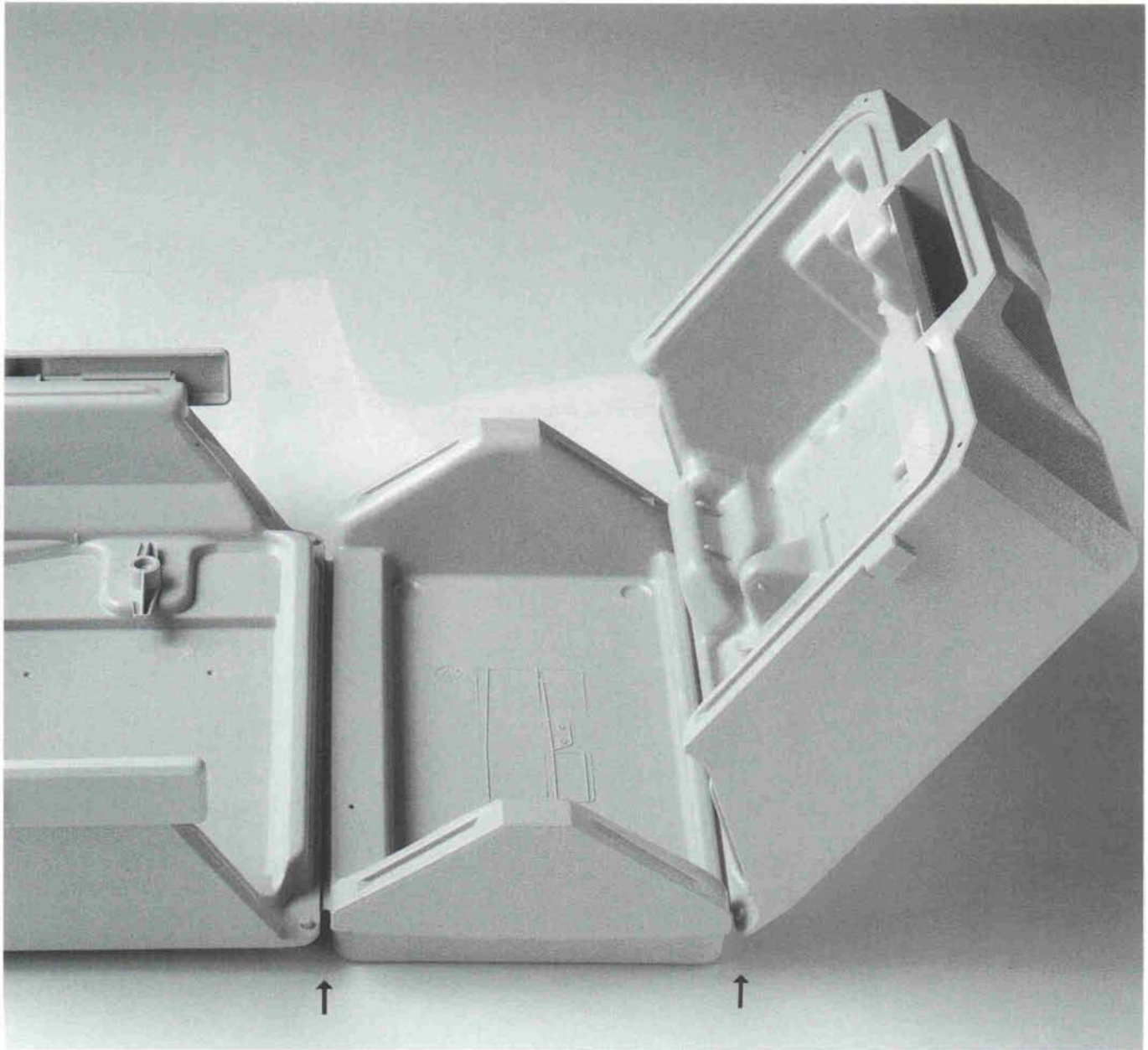




8.6 Transmission head of an electric razor

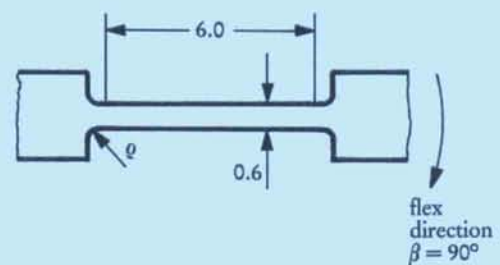
This transmission head made from Hostaform S 27063 has the function of converting the drive motion imparted by the drive motor into a reciprocation movement of the razor cutter. High oscillating frequency and a small flex angle characterize the stress to which the twelve integral hinges are exposed.

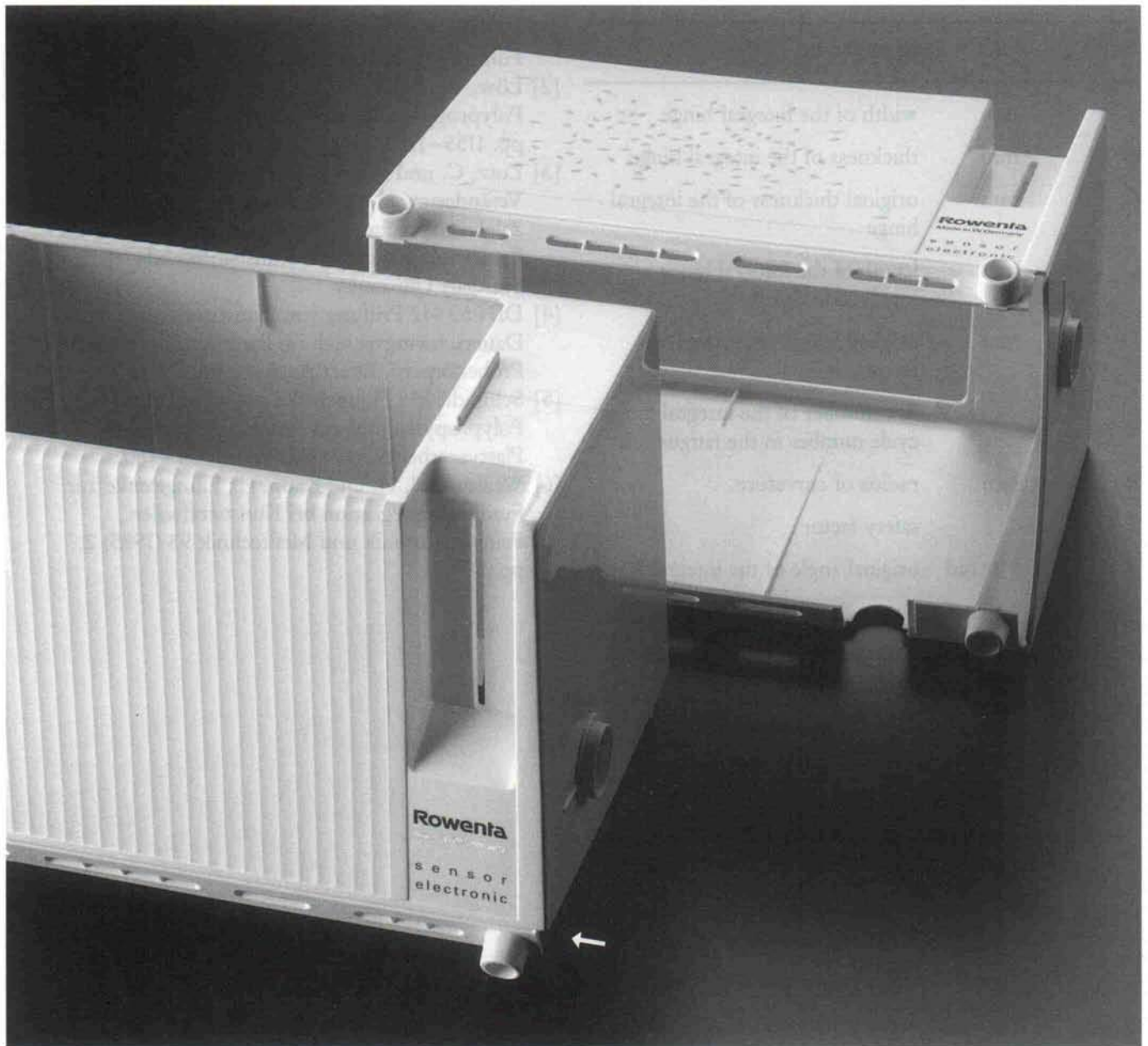




8.7 Sewing machine box

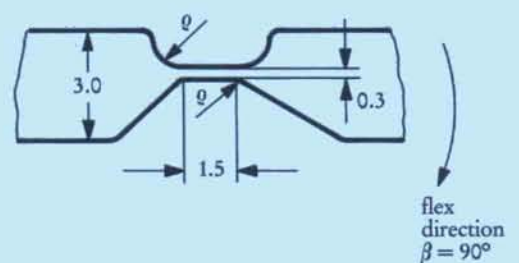
This box is extrusion blow moulded from Hostalen PPG 1022. To confirm the stress-bearing capacity of the integral hinge, flexing trials were carried out at room temperature and at -20°C . After 20 000 flex cycles at -20°C and $3 \cdot 10^6$ cycles at room temperature, no changes in the integral hinge could be detected except for slight white fracture.





8.8 Toaster housing

This toaster made from Hostacom M1 U01 is very easy to assemble. Two fixing strips with moulded-on feet are connected to the toaster body via integral hinges. When the toaster chassis has been inserted, the fixing strips are bent inwards through 90° and screwed to the chassis.



9. Explanation of symbols

Symbol	Unit	Explanation
b	mm	width of the integral hinge
h	mm	thickness of the integral hinge
h_0	mm	original thickness of the integral hinge
L	mm	length of the integral hinge involved in flexure
L_0	mm	original length of the integral hinge
N		flex number of the integral hinge cycle number in the fatigue test
R	mm	radius of curvature
S		safety factor
$\propto \alpha$	° or rad	original angle of the integral hinge
$\propto \beta$	° or rad	flex angle
ε_a		deformation amplitude in the fatigue test
ε_b		outer fibre deformation in the integral hinge
λ		degree of stretching $\lambda = \frac{h_0}{h} = \frac{L}{L_0}$
q	mm	fillet radius of the transition from the integral hinge to the moulding
σ_a	N/mm ²	stress amplitude in the fatigue test
σ_b	N/mm ²	flexural stress in the integral hinge

10. Literature

- [1] VDI/VDE 2252, Blatt 9: Feinwerkelemente; Führungen, Federgelenke (draft October 1987)
- [2] Löw, W.: Blasformen von technischen Teilen aus Polypropylen. *Kunststoffe* 78 (1988) 12, pp. 1155–1160
- [3] Lutz, C. und Polsack, A.: Morphologische Veränderungen von i-Polypropylen bei einachsiger Zugbeanspruchung. Mitteilung aus dem Inst. für Kunststoffprüfung und Kunststoffkunde (IKP), Stuttgart University
- [4] DIN 53 442 Prüfung von Kunststoffen, Dauerschwingversuch im Biegebereich an flachen Probekörpern. Draft August 1988
- [5] Schmidt, H.: Filmgelenke aus verstärktem Polypropylen und aus Acetalcopolymerisat. *Plastverarbeiter* 34 (1983) 9, pp. 774–780
- [6] Weißmantel, H. und Kapp, L.: Filmgelenke zur Funktionsintegration bei Kunststoffteilen. *Feinwerktechnik und Meßtechnik* 93 (1985) 2, pp. 89–91

Publications so far in this series:

A. Engineering plastics

- A.1.1 Grades and properties – ®Hostaform
- A.1.2 Grades and properties – ®Hostacom
- A.1.4 Grades and properties – ®Hostalen GUR
- A.1.5 Grades and properties – ®Celanex,
®Vandar, ®Impet
- A.2.1 Calculation principles
- A.2.2 ®Hostaform – Characteristic values and
calculation examples
- A.2.3 ®Hostacom – Characteristic values and
calculation examples

B. Design of technical mouldings

- B.1.1 Spur gears with gearwheels made from
®Hostaform, ®Celanex and ®Hostalen GUR
- B.2.2 Worm gears with worm wheels made from
®Hostaform
- B.3.1 Design calculations for snap-fit joints in
plastic parts
- B.3.2 Fastening with metal screws
- B.3.3 Plastic parts with integrally moulded threads
- B.3.4 Design calculations for press-fit joints
- B.3.5 Integral hinges in engineering plastics
- B.3.7 Ultrasonic welding and assembly of
engineering plastics

C. Production of technical mouldings

- C.2.1 Hot runner system – Indirectly heated,
thermally conductive torpedo
 - C.2.2 Hot runner system – Indirectly heated,
thermally conductive torpedo
Design principles and examples of moulds
for processing ®Hostaform
 - C.3.1 Machining ®Hostaform
 - C.3.3 Design of mouldings made from
engineering plastics
 - C.3.4 Guidelines for the design of mouldings
in engineering plastics
 - C.3.5 Outsert moulding with ®Hostaform
-

In these technical information brochures, Hoechst aims to provide useful information for designers who want to exploit the properties of engineering polymers such as ®Hostaform. In addition, our staff will be pleased to advise you on materials, design and processing.

This information is based on our present state of knowledge and is intended to provide general notes on our products and their uses. It should not therefore be construed as guaranteeing specific properties of the products described or their suitability for a particular application. Any existing industrial property rights must be observed. The quality of our products is guaranteed under our General Conditions of Sale.

Applications involving the use of the Hoechst materials ®Hostaform, ®Hostacom, ®Hostalen PP, ®Hostalen, ®Celanex and ®Vandar are developments or products of the plastics processing industry. Hoechst, as supplier of the starting material, will be pleased to give the names of processors of plastics for technical applications.



Hostaform[®], Celcon[®]
polyoxymethylene copolymer (POM)

Celanex[®]
thermoplastic polyester (PBT)

Impet[®]
thermoplastic polyester (PET)

Vandar[®]
thermoplastic polyester alloys

Riteflex[®]
thermoplastic polyester elastomer (TPE-E)

Vectra[®]
liquid crystal polymer (LCP)

Fortron[®]
polyphenylene sulfide (PPS)

Celstran[®], Compel[®]
long fiber reinforced thermoplastics (LFRT)

GUR[®]
ultra-high molecular weight polyethylene (PE-UHMW)

Россия

Руспласт ООО

Контактные данные

Тел.: +7-495-134-33-14

eMail: rusplast@rusplast.com

Сайт: www.rusplast.com

Russia

Rusplast LLC

Product Information Service

Tel.: +7-495-134-33-14

eMail: rusplast@rusplast.com

Internet: www.rusplast.com